

# Stepping, Strain Gating, and an Unexpected Force-Velocity Curve for Multiple-Motor-Based Transport

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## Summary

**Background:** Intracellular transport via processive kinesin, dynein, and myosin molecular motors plays an important role in maintaining cell structure and function. In many cases, cargoes move distances longer than expected for single motors; there is significant evidence that this increased travel is in part due to multiple motors working together to move the cargoes. Although we understand single motors experimentally and theoretically, our understanding of multiple motors working together is less developed.

**Results:** We theoretically investigate how multiple kinesin motors function. Our model includes stochastic fluctuations of each motor as it proceeds through its enzymatic cycle. Motors dynamically influence each other and function in the presence of thermal noise and viscosity. We test the theory via comparison with the experimentally observed distribution of step sizes for two motors moving a cargo, and by predicting slightly subadditive stalling force for two motors relative to one. In the presence of load, our predictions for travel distances and mean velocities are different from the steady-state model: with high motor-motor coupling, we predict a form of strain-gating, where—because of the underlying motor's dynamics—the motors share load unevenly, leading to increased mean travel distance of the multiple-motor system under load. Surprisingly, we predict that in the presence of small load, two-motor cargoes move slightly slower than do single-motor cargoes. Unpublished data from G.T. Shubeita, B.C. Carter, and S.P.G. confirm this prediction *in vivo*.

**Conclusions:** When only a few motors are active, fluctuations and unequal load sharing between motors can result in significant alterations of ensemble function.

## Introduction

Directed movement of cargoes along MTs is a key component of transport within a cell. Numerous studies suggest that cargoes *in vivo* are moved by more than one microtubule-based motor [1], but relatively little is known about the combined function of multiple motors. Our past experimental work established that multiple-kinesin motors can move cargoes long distances and that the force required to stall a bead moved on average by two kinesins was slightly less than double the force required to stall a bead moved by a single kinesin [2]. However, we would like more insight into how the motors work together: how do they step, how do they influence each other, how do they respond to an external force

that opposes their motion (load), and how is their ensemble function affected by the combination of thermal noise and viscosity?

Molecular motors are stochastic enzymes; in the small-*N* limit, fluctuations need not average out. To investigate how multiple interacting motors function, and whether fluctuations are important, we developed a Monte Carlo formalism extending a previous description of a single-kinesin motor (see [Supplemental Data](#) in [3]).

Although not relevant for an existing steady-state model [4], a key issue for the Monte Carlo simulation is the compliance of the linkage that connects the motor heads to the cargo ([Figure 1](#)). In principle, the magnitude of this linkage stiffness could vary significantly *in vivo* because it includes multiple sources of compliance that could exist linking the two motors, including stretching of the motors themselves, stretching of any scaffold proteins linking the motors to the cargo, and deformation/stretching of the cargo itself. The strength of the coupling between the motors (set by linkage stiffness) determines how information is propagated between them. We find that coupling strength plays a significant role in determining the ensemble's function—for weak coupling, fluctuations are not well communicated, and the system's performance approaches the steady-state model. In contrast, when stronger (likely realistic) coupling coefficients are used, the system's performance is quite different.

## Results

### General Procedure

We start with a model of single-kinesin motors that reproduces the existing [5] single-molecule measurements (see [Supplemental Data and Figures S1 and S2](#)). Motors could be placed randomly anywhere on a model cargo or could be placed together (clustered) at a single spot. We chose the latter geometry, based both on *in vivo* work indicating that this is likely the way motors are arranged on physiological cargoes [6] and on the desire to compare with an existing theoretical description that implicitly uses such a geometry [4]. We put  $N = 2, 3$ , or 4 motors on the cargo at a single spot ([Figure 1](#)) and allow the motors to bind to the microtubule stochastically with a probability based on their “on-rate.” Simulations typically start with a single motor attached to the microtubule. The motor(s) then walk along the microtubule, with each motor progressing through its kinetic cycle. Individual motors have a probability of detaching from the microtubule at each step, and conversely, at each time step unattached motors can reattach. The cargo continues along the microtubule, instantaneously driven by a number  $n$  of *engaged* motors ( $n$  is less than or equal to  $N$ , and is updated at every simulation step in accord with the motors' dynamics), until the simulation ends, or  $n = 0$  indicating all motors have fallen off the microtubule.

Each motor attaches to only the bead and the microtubule; there is no direct motor-motor interaction. However, motors can influence each other through the bead. For instance, a stationary motor ([Figure 1](#), brown motor) can exert a backward force on a moving motor (blue motor) through the cargo. The magnitude of such effects is determined by the linkage

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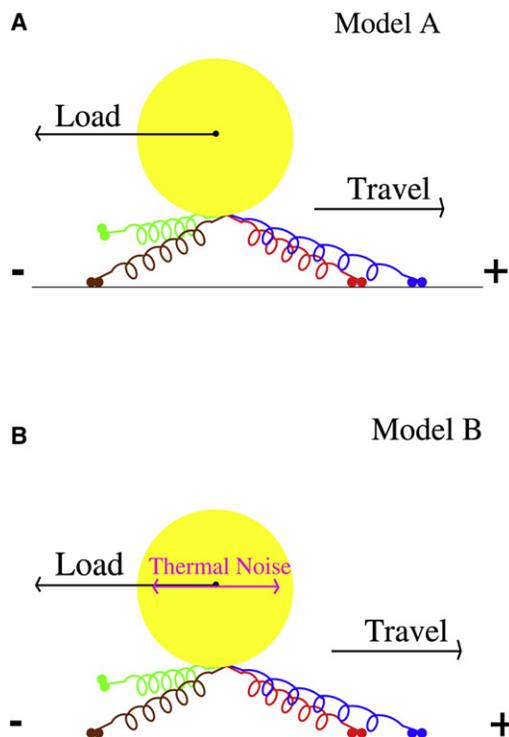


Figure 1. Diagram of Model A and Model B

The cargo (yellow circle) is attached to four motors (colored “springs” with two small blobs at their end, indicating the motors’ heads). The motors can attach to and detach from the microtubule (black line). In the case shown, although there are  $N = 4$  motors attached to the cargo, there are only  $n = 3$  engaged motors, because the green motor is not bound to the microtubule. Although dynamics determine the instantaneous number of engaged motors  $n$ , depending on the simulation, there is an initial choice of  $N$  between 1 and 4 motors.  $N$  does not vary once a simulation starts. The main difference between models A (A) and B (B) is that model B includes thermal noise providing “kicks” in random directions and the viscosity of the medium. Details of each model are provided in [Supplemental Data](#).

stiffness. If motors are flexible and easily extended, the moving motor(s) advance with relatively little opposition; if the motors are stiff, the unmoving motor can in principle significantly inhibit motion. In practice, each motor is modeled as a spring of natural length  $l$  ( $l = 110$  nm) that exerts a restoring force only when stretched beyond  $l$ . The spring (motor) has no compressional rigidity, i.e., it buckles without resistance when compressed. This leads to strong or weak coupling among motors, for high or low linkage stiffness, respectively.

The experimental readout for transport (both in vivo and in vitro) is the position of the cargo, not the locations of individual motor heads on the microtubule. Therefore, the position of the cargo must be calculated at each step of our simulation, taking into account the motors’ positions and spring constants and any externally applied load. We considered two related models, models A and B (Figure 1). Both include the stochastic nature of the motors’ enzymatic cycles and their coupling, but the more general model (Figure 1, model B, bottom) also incorporates the effects of thermal noise and viscosity of the medium on the cargo’s motion. Both models are one-dimensional, so that forces, velocities, and displacements are along the direction of the microtubule. The models are complementary: model B is more realistic, and used for specific predictions, but the simplicity of model A provides additional

insight into the underlying factors leading to the deviations in system performance found in our models in comparison to continuum descriptions of motion [4]. Details of both models’ implementation are provided in the [Supplemental Data](#). In contrast to the continuum theory, these models make no assumption about equal load sharing. We find that the motors’ dynamics themselves frequently result in unequal load sharing and in load-induced coordination.

#### Prediction of Step-Size Distributions

Individual kinesin heads move in 16 nm advances [7], resulting in 8 nm advances of the motor’s center of mass [8]. However, it is not well understood how a cargo’s center of mass advances when moved by two or more motors: do motors synchronize, moving the cargo in increments of 8 nm, or do they function independently, so that the center of mass moves in 4 nm steps? This question is relevant for interpreting in vivo studies that observed 8 nm steps [9, 10]: could these steps reflect transport driven by multiple motors, or when 8 nm steps occur, do they reflect only single-motor function? Model A predicts that under a low to medium load ( $\sim 3$  pN), the cargo’s center of mass should almost all of the time advance in 8 nm steps (Figure 2A). This reflects two features. First, the force-velocity curve of individual motors is insensitive to load at low load (Figure S1A), so the motors need not share load—although the forward motor supports most of the load, it advances at approximately the same rate as the rear motor, so the motors’ microtubule attachment locations can be well separated (Figure S5). Thus, applied load is supported by the forward motor, and this motor determines the cargo’s position: when it steps by 8 nm, the cargo advances 8 nm; the rear motor advance does not effect the cargo’s position. However, this prediction is inconsistent with experimental data [11]—cargoes driven by single-kinesin motors do advance in 8 nm increments but this is not the case for cargoes in buffer driven predominantly by two motors (see experimental data in Figure 2B).

Importantly, model A neglects Brownian noise: in the absence of such noise, applied load moves back the cargo’s center of mass, and its position is determined predominantly by the forward motor. However, when Brownian noise is present, thermal fluctuations allow the cargo to sample enough positions that it “sees” the rear motor. In this case, the cargoes’ position reflects applied force and both motors’ locations. Model B includes such effects, and agrees well with experiment (Figure 2B) except for small steps (0–1.5 nm). Unlike the experimental analysis [11], we did not join closely spaced steps after using the step detection algorithm on the simulated data, because we had a priori knowledge of the number of steps in the record. Such joining suppresses small steps, likely accounting for the decrease in small steps in the experimental record relative to the theoretical data (Figure 2B).

For this comparison of theory and experiment, there were essentially no free parameters. Although model B correctly predicts the cargo’s displacements and model A does not, this does not necessarily imply that the activity of the individual motors themselves are different in the two models. In fact, the distribution of distances between the locations of where the two motors bind to the microtubule is quite similar in the two models (Figure S5, compare A versus B). Thus, much of the difference in predicted step distributions between the two models likely reflects the effect of Brownian noise on our “indicator” of motor function—the position of the cargo—rather than differences in motor activity itself.

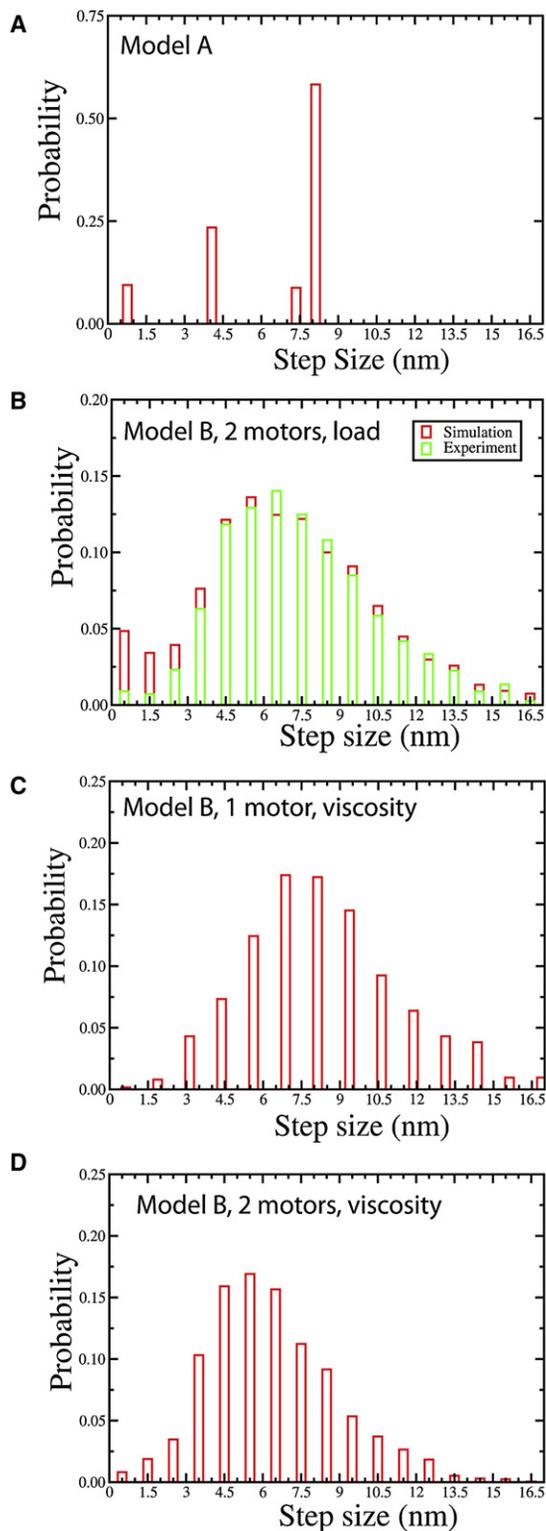


Figure 2. Predicted Motions of the Center of Mass of the Cargo

(A) Predicted distribution of step-wise advances of the cargo, resulting from two motors moving a cargo under low load, according to the dynamics of model A.  
 (B) Predicted distribution of step-wise advances of the cargo, resulting from two motors moving a cargo under low load, according to the dynamics of model B. Here the cargo's motion was simulated, according to model B, and then the resulting displacement records were analyzed exactly as for experimental data, by the step-detection method previously investigated

Having experimentally confirmed model B by predicting the experimental distribution of step sizes, we investigated the important but less experimentally tractable situation of transport in cells where the effective cytosolic viscosity could be many times that of water. In principle at very high viscosities, the cargo's thermal motions might be slow enough that it would not "see" the rearward motor, and so the cargo's motion would then show step-like advances of  $\sim 8$  nm, as predicted by model A. We investigated this issue with model B, where we applied no external load, but assumed a spherical (0.5 micron diameter) cargo moving through a solution with viscosity  $\sim 100\times$  that of water (consistent with the maximum viscosity likely in the cytosol [G.T. Shubeita, B.C. Carter, and S.P.G., unpublished data]). For a cargo driven by a single kinesin, we predict 8 nm steps (Figure 2C). For a cargo moved by two kinesins under the same conditions, the distribution of detected step sizes in simulated data (Figure 2D) is similar to that in buffer under low load (Figure 2B), suggesting that high viscosity of the cytosol by itself is unlikely to result in 8 nm steps for cargoes moved by two motors. At higher values of viscosity ( $\sim 1000\times$  water), it becomes difficult to detect clean steps (because the step rise time is slowed), but to the extent that we detect them, we observe the same type of behavior as in Figure 2D (not shown). Thus, the observed 8 nm steps in vivo likely reflect either cargoes moved by single motors or cargoes moved by two unsynchronized motors each undergoing 16 nm steps [10, 12], or they reflect some synchronization of motors that occurs in vivo but not in vitro.

#### Testing the Theory: Two-Motor Stalling Forces

We also tested our model by predicting the force required to stall a cargo moved by two versus one motor. The unitary stall force is a free parameter; to compare with our experiments, we set the one-motor stall force to be 4.8 pN. The other adjustable parameter is the motor "on" rate (chosen to be 5/s); this affects the observed number of one-motor events versus two-motor events (not shown). The compliance can significantly affect how the motors work together (see below), but is not a free parameter—the relevant value (used here) was experimentally determined to be 0.32 pN/nm for single kinesin-1 motors in vitro [13]. With this choice of values, we then did in silico experiments: simulations were done for two-motor beads moving in an optical trap of appropriate stiffness, according to the dynamics of model B. Such simulations generated

[11]. The distribution of detected steps (red histogram bars) are compared to previous experimental data (green bars) from [11].

(C) Predicted distribution of step-wise advances of the cargo, resulting from a single motor moving a  $0.5 \mu\text{m}$  cargo under no load, but in the presence of a viscosity  $\sim 100\times$  that of water (viscosity of water = 0.001 Pa-S), according to the dynamics of model B.

(D) The same as (C), but for two motors. The parameter values are  $k_{on} = 2 \times 10^6 \text{ M}^{-1} \cdot \text{s}^{-1}$  and  $k_{cat} = 105 \text{ s}^{-1}$  (see Equation 1 in Supplemental Data),  $K_{O\ off} = 55 \text{ s}^{-1}$  and  $d_1 = 1.6 \text{ nm}$  (see Equation 2.4 in Supplemental Data),  $[\text{ATP}] = 3 \text{ mM}$  and  $B = 0.029 \mu\text{M}$  (see Equation 3 in Supplemental Data),  $A = 107$  and  $\delta_1 = 1.3 \text{ nm}$  (see Equation 4 in Supplemental Data), and  $F_o = 5 \text{ pN}$ . These parameters except  $F_o$  are same used in Figures S1 and S2 that fit extremely well to the experimentally measured single-molecule data. The overall length of each motor  $l$  is chosen to be  $l = 110 \text{ nm}$  with compliance  $k = 0.32 \text{ pN/nm}$ . The "on" rate ( $P_a$ ) for each motor was assumed to be  $5 \text{ s}^{-1}$  and the rate of detachment under a load equal to or higher than stall  $F_o$  is  $P_{back} = 2 \text{ s}^{-1}$ . These parameter values are used throughout the manuscript unless indicated otherwise;  $F_o = 5 \text{ pN}$  (used in the simulation here) is a "tunable" parameter and is chosen to match our experiments; there are no other free parameters.

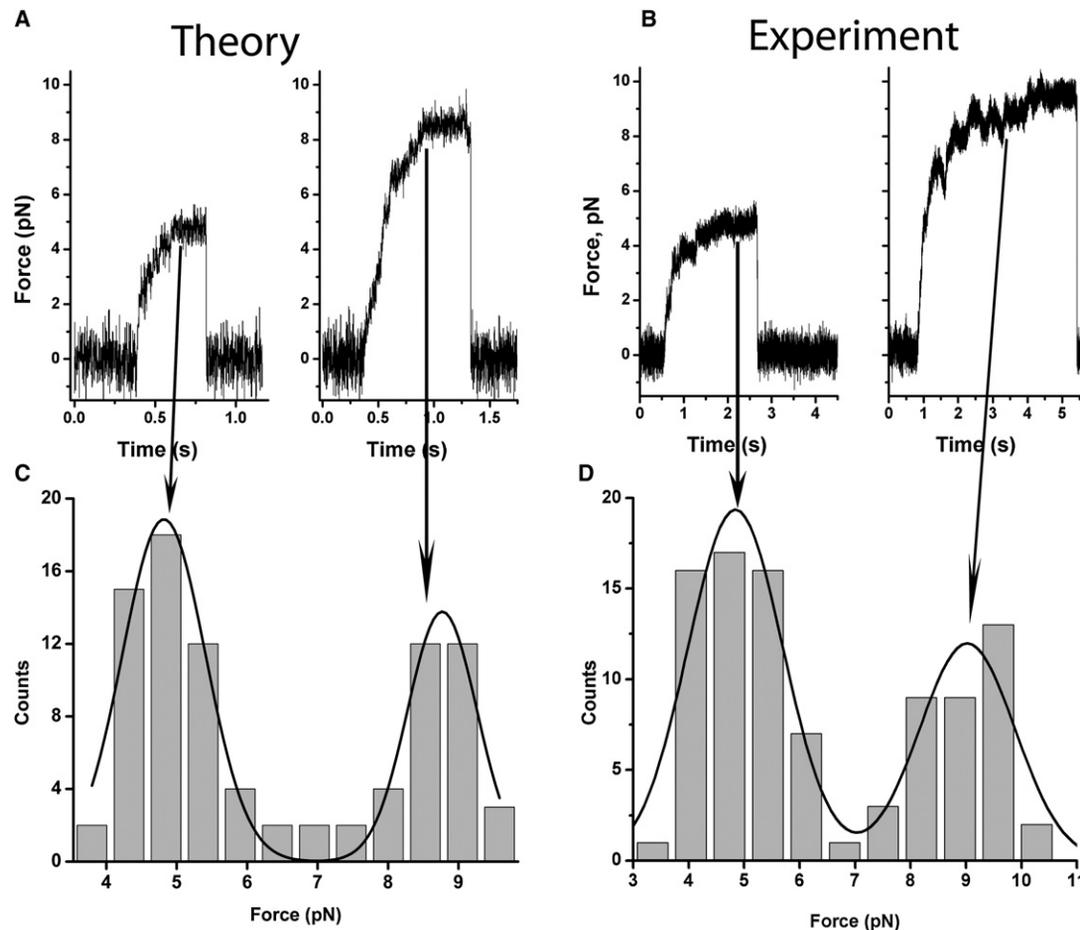


Figure 3. Predicted Distributions of Stall Forces for Cargoes Moved by One and Two Motors

Examples of traces of the cargo's motion as it moves in an optical trap, for simulated cargoes moving according to the dynamics of model B (A), or actual beads moved by purified kinesin motors in vitro (B). In each case, the left trace is a "one-motor" stall and the right trace is a "two-motor" stall. By eye, the simulated and experimental stalls appear quite similar. Multiple such traces, both simulated and experimental, were taken and analyzed according to our standard procedure [2] to yield the distribution of stalls predicted by model B (C) or experimentally observed (D). The only free parameter for the theory was the magnitude of the one-motor stall, which was tuned to yield observed stalls at approximately 4.8 pN (to match the experimentally determined one-motor stalls). For the theory, the fits of the gaussians (C) yield peak locations of  $4.82 \pm 0.05$  pN and  $8.77 \pm 0.07$  pN. For the experiments (D), the peaks are at  $4.8 \pm 0.1$  pN and  $9.0 \pm 0.2$  pN.

With the exception of  $F_0$  ( $= 2.8$  pN), the parameter values are same as used in Figure 2.

simulated stalling-force records of bead displacement as a function of time, which by eye resembled similar traces from actual experiments (Figures 3A and 3B). Such simulated traces were analyzed with the same criteria to detect and quantify stalls as for experimental data. The histogram of detected stall forces (Figures 3C and 3D) shows two distinct peaks (Figure 3C) centered at  $4.82 \pm 0.05$  pN and  $8.77 \pm 0.07$  pN, whereas for the experimental data (Figure 3D), the peaks are centered around  $4.8 \pm 0.1$  pN and  $9.0 \pm 0.2$  pN. Because the one-motor stall is a tunable parameter in our model, for theory-versus-experiment comparison, we use the two-motor to one-motor stall force ratio. We theoretically predict a ratio of  $1.820 \pm 0.025$ ; the experimental ratio for kinesin-1 motors is  $1.875 \pm 0.066$ . Thus, the simulations and experiments are consistent.

The best way to compare theory and experiment is to perform data analysis identically (as above), but this is impossible with the continuum theory, which describes only the steady-state average behavior of the motors. The challenge is to predict what is actually measurable: a "stall" occurs when the

mean velocity is zero, so it is tempting to use the theories' force-velocity curves (e.g., Figures 4A–4C) to predict the force at which the mean velocity is zero. However, such a force is frequently experimentally unobservable because the mean travel becomes minimal before such a force is achieved. For instance, both the continuum theory and our model B predict that for two  $\sim 6$  pN stall motors, the velocity goes to zero at 12 pN of externally applied load (Figures 4B and 4C, red curves), but the theories also predict that the mean travel becomes  $\sim 1$  step of the motors at 8.8 pN (continuum, dotted red curve, Figure 5B) or 10.4 pN (model B, solid red curve, Figure 5B). According to a suggestion in [4], we use the theoretical force-persistence curves (see next section) to identify the approximate location of the stalls, by determining the force for which a cargo's mean travel distance goes to  $\sim 8$  nm (one step). With our simulations, we evaluate how well this approach approximates the in silico experiments above.

One complication for this comparison is that the kinetic properties of individual motors must be tuned differently, depending on whether we aim to observe the mean one-motor

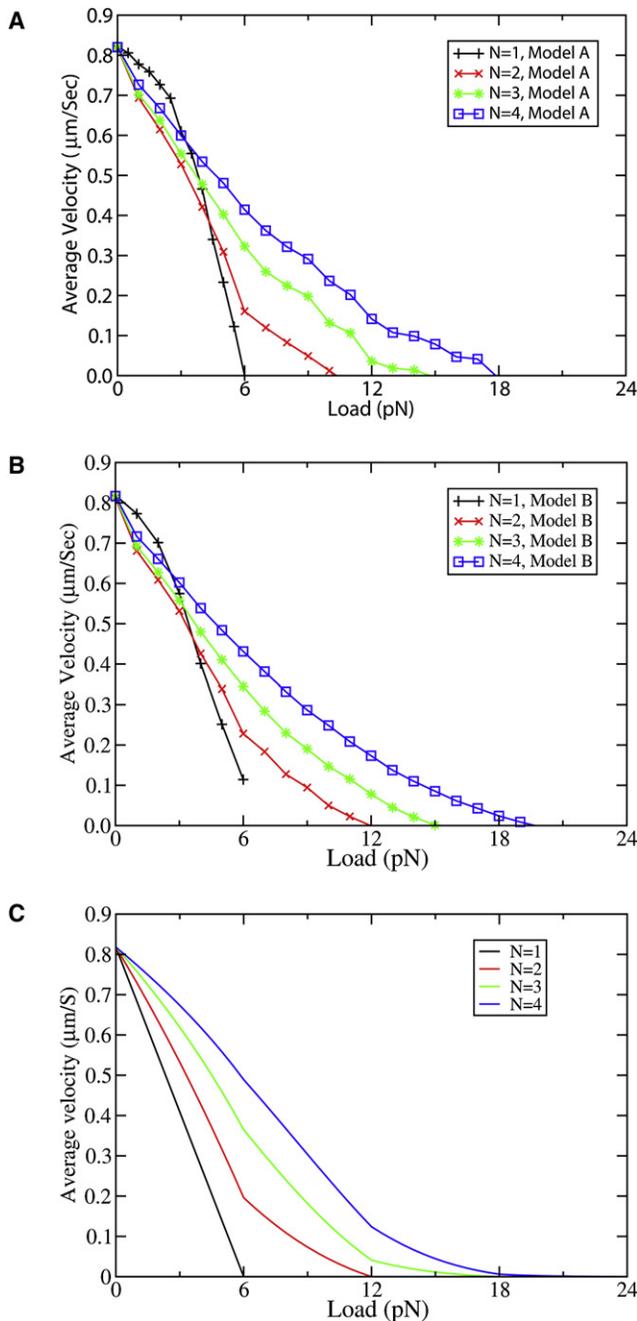


Figure 4. Force-Velocity Curves

(A and B) Predicted force-velocity curves from our multiple-motor model for  $N = 1, 2, 3,$  and  $4$  motors from model A (A) and model B (B), assuming a linkage stiffness of  $0.32 \text{ pN/nm}$  and an “on” rate of  $P_a = 5 \text{ s}^{-1}$ . The parameter values and initial conditions are the same as in Figure 5A. The average velocity was calculated with the velocities calculated over a time window of  $0.5 \text{ s}$  in the steady state.

(C) The force-velocity curves from the steady-state (SS) model, also assuming an “on” rate of  $5 \text{ s}^{-1}$ , a single-motor stall force of  $6 \text{ pN}$ , and identical single-motor velocities. The parameter values chosen for SS model are  $v = 0.818 \text{ µm/s}$ ,  $\pi_{ad} = 5 \text{ s}^{-1}$ ,  $\epsilon_n = 1 \text{ s}^{-1}$ ,  $F_d = 3 \text{ pN}$ , and  $F_s = 6 \text{ pN}$  (see Equations 7, 8, and 10–12 in Supplemental Data).

stall at  $4.8 \text{ pN}$  via either of the two approaches. From a practical point of view, the difference is irrelevant because the ratio of the two-motor to one-motor stall is approximately constant, independent of the absolute value of the one-motor stall. If

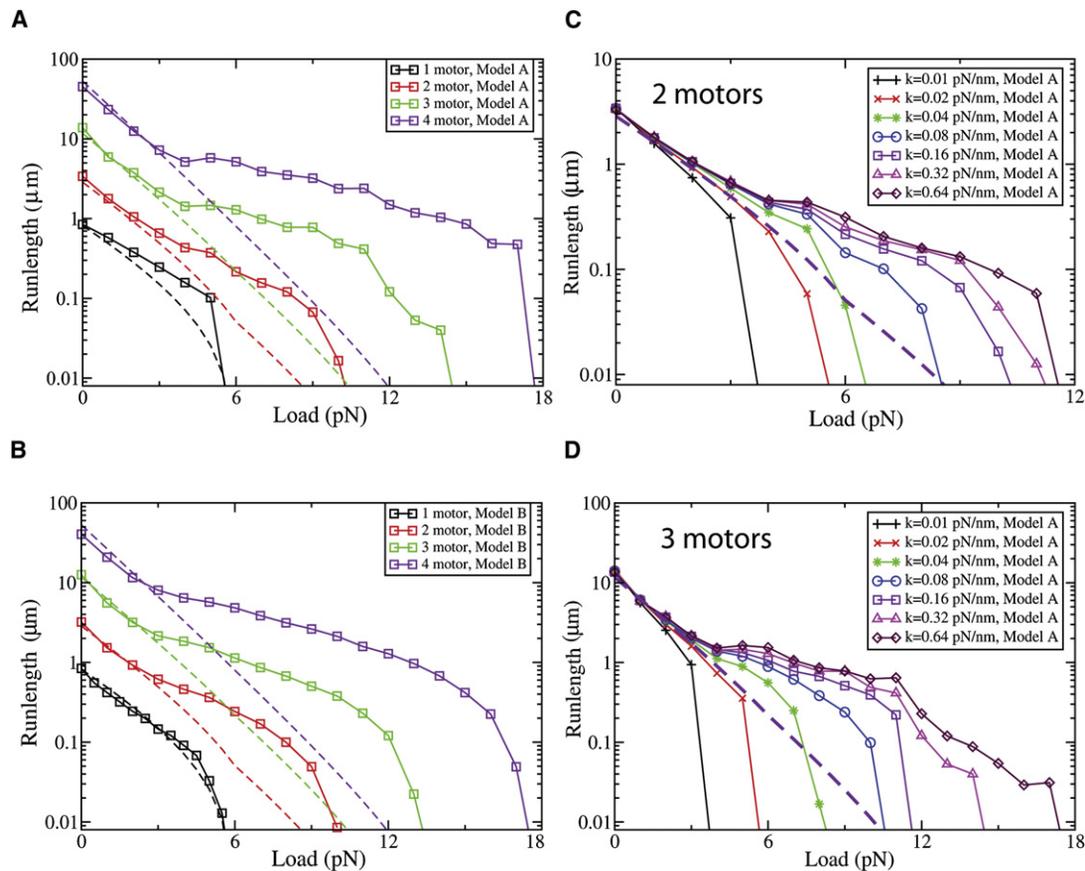
we tune the one-motor stall to be  $4.8 \text{ pN}$  (according to the mean travel distance criteria), the approximate location of the two-motor stall is predicted to be  $8.5 \text{ pN}$  (Figure S4), or a two-motor to one-motor ratio of  $1.77 \pm 0.03$ . In absolute terms, the value of  $8.5$  is about  $0.27 \text{ pN}$  less than the peak location determined above by the in silico approach, reflecting roughly a 3% error in determining the theoretical stall value from the two different methods. This fast method to compare model B with experiments thus yields a theoretical prediction that is off by approximately  $0.5 \text{ pN}$ , or 5.5%. In contrast, after tuning the continuum model parameters [4] to have a one-motor stall of  $4.8 \text{ pN}$ , the two-motor stall is predicted to be  $7.4 \text{ pN}$ , low by  $1.6 \text{ pN}$  or 17.8% relative to experiments (Figure S4, dotted red line). The continuum theory thus predicts a two-motor to one-motor ratio of  $1.54 \pm 0.04$ , in contrast to our predicted ratio of  $1.77 \pm 0.03$ , and the experimentally measured ratio of  $\sim 1.87$ .

### The Force-Persistence Curve

Molecular motors are cargo transporters, so it is important to characterize how far and how fast they transport a cargo, and how this is affected by opposition to motion (applied load or viscous drag). For single motors, these effects are summarized by the force-processivity (Figures S1C, S1D, and S2B) and force-velocity (Figures S1A, S1B, and S2A) relationships that measure the motor’s average travel distance and velocity as a function of applied load (force opposing the motor’s advance). We extended these ideas to characterize how multiple motors move cargoes. We consider individually processive motors, making this very different from previous work on linkage stiffness effects on ensembles of single-headed non-processive motors [14]. We first determined the force-persistence curves for cargoes carrying  $N = 2, 3,$  and  $4$  motors (Figures 5A and 5B). The coupling between the motors significantly affects the ensemble performance (Figures 5C and 5D), so the force-persistence relationship for a given  $N$  is not unique, but is a function of motor stiffness. In the simulations below (Figures 5A and 5B), we assume the individual motor stiffness to be  $0.32 \text{ pN/nm}$ , the in vitro [13] stiffness of a kinesin-1 motor.

At no load, the predicted run lengths from either model A or B (Figures 5A and 5B, symbols and solid lines) are quantitatively in good agreement with the previous [4] steady-state description (Figure 5, dotted lines of same color), for  $N = 1, 2, 3,$  and  $4$  motors. However, the predictions diverge (compare solid and dotted lines of the same color) from the continuum model when the motors function under load (Figures 5A and 5B).

When load is present, the magnitude of the coupling between motors (determined by the spring constant  $k$  characterizing the compliance of the linkage, see Figure 1) has large ramifications on system behavior. As the stiffness of the linkage between the motor heads and the cargo increased, so did average cargo travel under constant load (Figures 5C and 5D, for cargoes with 2 and 3 motors, respectively; similar curves for model B are found in Figure S7). Two effects present in these models but not in continuum models account for these observations. First, for a low-stiffness linkage, when one motor detaches, the applied load supported by the remaining motor stretches the linkage, resulting in load-induced backward motion of the cargoes’ center of mass. Second, when the motors have high-stiffness linkages, under high load they dynamically alternate between stepping and supporting load. Each of these effects is discussed below.



**Figure 5. Force-Persistence Curves for Cargoes**

For comparison with the continuum theory [4], simulations were done for motors with a single-motor stall tuned to  $\sim 5.7$  pN. All models assumed an “on” rate of  $5 \text{ s}^{-1}$  and an identical single motor unloaded velocity.

(A and B) The linkage stiffness was fixed ( $0.32 \text{ pN/nm}$ ), but there were different total number of attached motors ( $N$ ). Curves shown are the predicted mean travel distance of 1, 2, 3, and 4 motors under different degrees of opposing force, moving according to model A (A) or model B (B). The dotted line reflects the predicted curve from the steady state model (SS) of Klumpp et al. [4]. The Monte Carlo simulations (MC) started with the condition of all motors initially attached to microtubule. The parameters chosen for Monte Carlo simulations of model A and model B are the same as in Figure 2, except  $F_0$ , which was tuned to stall of  $\sim 5.7$  pN;  $F_0 = 6 \text{ pN}$  and  $F_0 = 5.1 \text{ pN}$  for models A and B, respectively. The parameter values chosen for SS model are  $v = 0.818 \text{ } \mu\text{m/s}$ ,  $\pi_{ad} = 5 \text{ s}^{-1}$ ,  $\varepsilon_n = 1 \text{ s}^{-1}$ ,  $F_d = 3 \text{ pN}$ , and  $F_s = 6 \text{ pN}$  (see Equations 9–12 in Supplemental Data).

(C and D) Force-persistence curves for cargoes at different values of linkage stiffness, for two (C) and three (D) motors moving according to model A. The dotted black line is the prediction of the SS model [4], for the same parameters as in (A) (chosen to achieve agreement for the one-motor curve), for two motors (A) and three motors (B). All parameters chosen for the MC simulation of model A are the same as in (A), except for different  $k$  values as indicated. The MC initial conditions were the same as in (A), and the steady-state models’ parameter values were also as in (A). The related plot for model B is found in the Figure S7. The velocity of single motor for SS model was chosen to be  $0.818 \text{ } \mu\text{m/s}$  to match the single-motor Monte Carlo simulation.

### Impaired Function at Low Stiffness because of Stretching the Linkage

Cargoes with low stiffness motors on average travel shorter distances under load. When one motor detaches, the applied load supported by the remaining attached motors stretches their linkages and produces a backward displacement of the cargo’s center of mass; the higher the stiffness  $k$ , the less linkage stretching, and the less backward travel (Figure S8A). Interestingly, the motors most likely to detach are the “vanguard,” i.e., those that are farthest along the microtubule and that bear the most load. When such a motor detaches and the cargo moves backward, the unengaged motors likely reattach behind the original vanguard location. Indeed, our modeling predicts that the more the cargo’s center of mass moves back, the less likely it is to successfully return to its predetachment position (Figure S8B).

### Improved Function at High Stiffness because of Dynamic Strain Gating

The backward motion of the cargo resulting from the stretching linkage, combined with the failure to recover after detachment, explains the impaired performance relative to the steady-state model. What accounts for the improved performance at high stiffness? Consider, for example, two engaged motors with high spring constants, with a backward load of  $F$  applied to the cargo. When one motor advances ahead of the other by  $8 \text{ nm}$  or more, it supports significantly more than  $F/2$  of the applied load (in the limit of an infinitely stiff linkage, it supports the entire load  $F$ ). We hypothesize that this leads to strain gating: when a motor is ahead, it supports a high load, but is unlikely to step (the applied load decreases its velocity, i.e., decreases the mean stepping rate). In contrast, the rear motor is under low load (most load is supported by the forward motor),

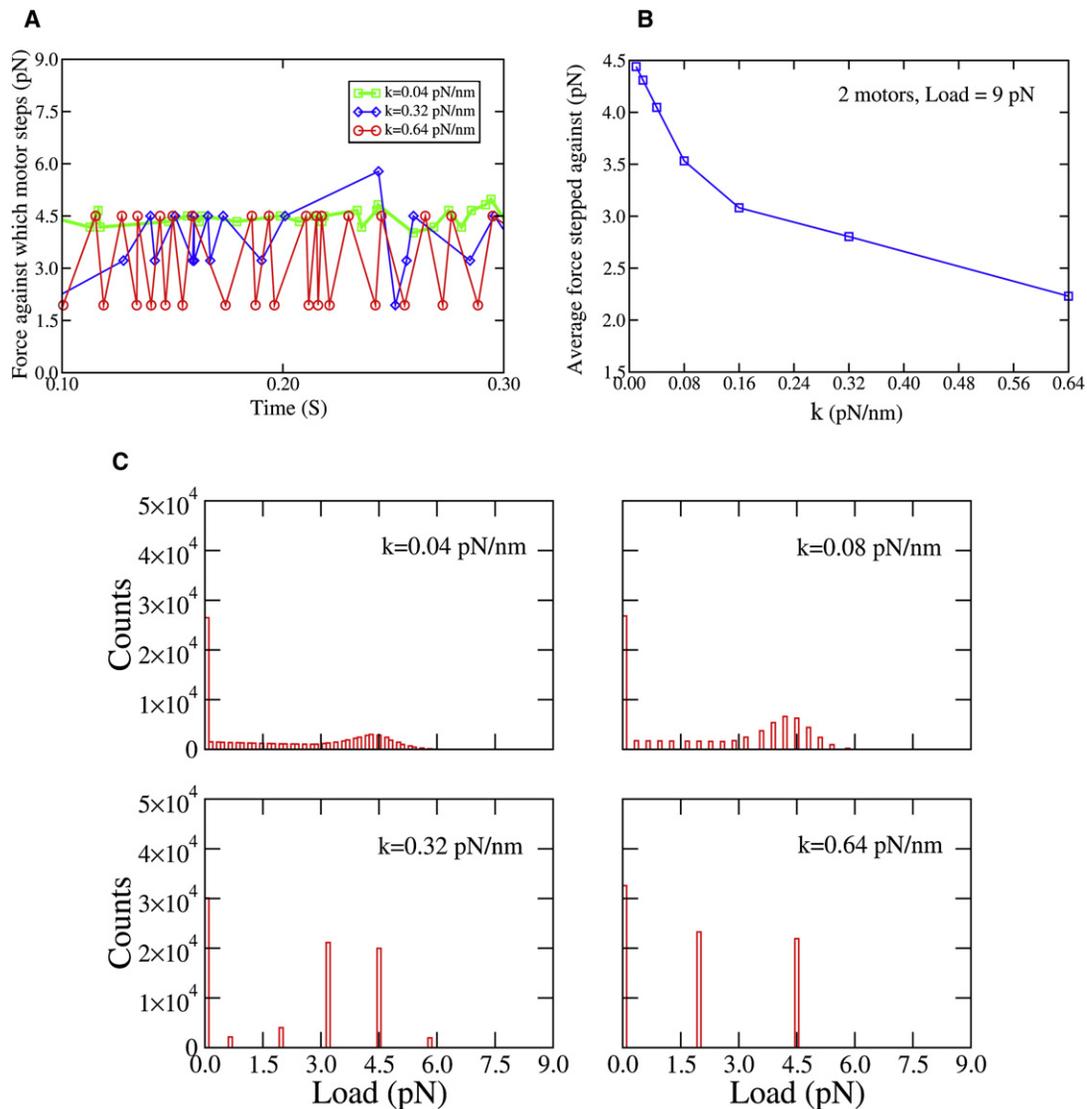


Figure 6. Strain Gating Explored for a Cargo Moved by a Maximum of  $N = 2$  Motors

(A) A trace of forces against which motors step, for different values of  $k$ , when the cargo experiences an applied load of 9 pN. The simulation was started with initial condition that both motors are attached randomly to the microtubule and simulation was stopped when one of the motors detached.

(B) The average force that a motor attached to the cargo steps against in model A, when the cargo experiences 9 pN of externally applied load, as a function of the stiffness of the linkage connecting the motors to the cargo. Initial and final conditions are identical to (A). The related curve for model B is in Figure S10.

(C) The distribution of forces the motors stepped against when a load of 9 pN was applied to the cargo. The four panels show the distribution for steps for motors linked to the cargo by different spring constants. Initial conditions identical to (A). All parameters chosen for MC simulations in this figure are same as in Figure 5C.

and thus steps faster, catching up with the forward motor. When both motors are at the same position, they are equally likely to step, and one steps against  $F/2$ . Thus, a typical scenario for two stiff engaged motors under high total load  $F$  is that every other step is against a load less than  $F/2$ , so relative to a continuum model where the motors always share load equally, the mean force that a given motor steps against is reduced. Note that “strain gating” as used here differs from the previous use describing intermolecular interactions between kinesin heads (see Supplemental Data).

We tested this hypothesized strain gating in a number of ways. First, for a two-motor cargo (with 5.8 pN stall motors, as in Figures 5A and 5B) moved against 9 pN of externally applied load, we used model A to directly observe the sequence of forces that were stepped against (Figure 6A). For strain-

gating, we expect an alteration of forces, with a step against  $F/2$ , then one against a lower force, then against  $F/2$ , etc. This sequence of stepping (Figure 6A) was observed. For lower stiffness cases (e.g.,  $k = 0.08$  pN/nm, green curve), where strain gating is not expected, each step should occur at approximately  $F/2$ . This too was observed (Figure 6A), and steps against a force of more than  $F/2$  were sometimes observed, corresponding to the forward motor stepping repeatedly. Such steps are not present when perfect strain gating occurs ( $k = 0.64$  pN/nm, Figure 6A, red curve) because the forward motor does not step. For a relatively high, but still intermediate, stiffness ( $k = 0.32$  pN/nm, blue curve), strain gating occurs but is not perfect, so although most of the time the  $F/2$ , lower than  $F/2$ , sequence occurs, occasionally a step occurs at larger than  $F/2$ .

The example traces (Figure 6A) are supported by the overall distribution of forces stepped against, for two motors moving a cargo against 9 pN of externally applied load (Figure 6C). In the low-stiffness case ( $k = 0.04$  and  $k = 0.08$  pN/nm, Figure 6C, top two panels), the forces stepped against are clustered around  $F/2$  (i.e., 4.5 pN). In the high-stiffness case, there are two peaks reflecting steps of the rear motor (at less than  $F/2$ ), and either of the motors (against  $F/2$ ) when they are side-by-side, and sharing load equally. In all panels, the steps against low loads (less than about 1.5 pN) can be ignored because they reflect a transient case where one of the motors has detached, and then reattached significantly behind its initial position, so that it feels little force after it reattaches, until it approaches its original position.

The benefit from strain gating depends on the linkage stiffness—for both  $k = 0.32$  and  $k = 0.64$ , the typical “high load” advance occurs at  $F/2$ , but the force that the rear motor steps at is lower for  $k = 0.64$  (peak is at  $\sim 1.8$  pN, Figure 6C, bottom right) than for  $k = 0.32$  (peak is at  $\sim 3.1$  pN, Figure 6C, bottom left) because in the higher stiffness case, the forward motor supports more of the load. Thus, we expect that the “average” force that a motor steps against will decrease, as a function of the linkage stiffness, as the forward motor supports increasing amounts of load, but does not step against them. This is observed in our simulations for both models (Figure 6B and Figure S10). A consequence of this decreased load should be to improve mean cargo travel, because a single motor’s processivity is a function of the load it steps against. This was observed (compare our predicted transport to the continuum model that assumes equal load sharing, Figures 5A and 5B).

Another consequence of applying high load/strain gating is to induce clustering of the motors on the microtubule. At high load, when the forward motor supports significant load, its enzymatic cycle is slowed (see force-velocity relationship for a single motor in Figures S1A and S2A [5]), resulting in the rearward motor(s) catching up. Thus, as the load increases, the average distance between the forward motor and rearward motor decreases. This is observed in both our models (Figures S5A and S5B). Thus, the “vanguard” motor in this case is unlikely to be far ahead of the rearward motor and if it were to detach then, the foremost microtubule-attachment point would not move back appreciably nor would the linkage stretch much. The net result is negligible rearward excursions of the cargo resulting from motor detachment. Note, however, that strain gating and the associated clustering happens only at high stiffness; for lower stiffness linkages, the motors do not cluster in the same way (Figures S6A and S6B).

We discussed strain gating in the simplest case of two motors, but the uneven load sharing between motors, combined with preferential stepping of the unloaded motors, occurs when more motors are present as well and leads to improved performance under load when the motors are coupled to the cargo via stiff linkages; this was observed for three motors as well (Figure 5D and Figure S7B).

### The Force-Velocity Curve

We turn now to the velocity behavior of a cargo carrying  $N = 2, 3,$  or  $4$  motors, as a function of applied load or viscosity. Both models provide similar curves (Figures 4A and 4B), which differ from the force-velocity curves predicted from the steady-state model (Figure 4C), particularly in the low-force regime (i.e., at an applied load less than or equal to the stall force of a single motor). For the steady-state model (Figure 4C), whenever load

is applied, the velocity for cargoes moved by multiple motors is higher than for cargoes moved by a single motor (as more motors share the load, the load per motor drops and motor velocity goes up). In contrast, our model predicts that at low loads, the velocity is *more* sensitive to load than in the single-motor case (black curve). This was unexpected because one motor is always attached (our simulations end when all motors detach).

We hypothesized that this unexpected dependence of velocity on load occurred because of detachment of some motors from the microtubule. To test this idea, we modified our simulation and did not allow motors to detach. Then, our result agrees with what is expected when motors share load equally all the time (Figure S11), and where the applied load does not affect the mean number of engaged motors—the two-, three-, and four-motor force-velocity curves simply involve rescaling the  $x$  axis by  $F/2$ ,  $F/3$ , and  $F/4$ , respectively, to reflect the lower load per motor.

What does motor detachment do? Consider two initially engaged motors. If they move together and share the load, we know (from the “no detachment allowed” simulation) that they will move faster than a single motor. If one motor detaches, the remaining motor will still not go slower than single motor. This logic also applies for more than two motors initially engaged, and it therefore appeared unlikely that the velocity sensitivity at low load resulted simply from changes in the mean number of engaged motors. Rather, we hypothesized that reduced velocity reflected backward motions of the center of mass of the cargo after a “vanguard” motor detachment, a mechanism discussed above (see also Figure S8) and illustrated in a cartoon (Figure 7C). Further discussion of this general mechanism is found in the Supplemental Data.

From this hypothesis emerge two predictions. First, there should now be a link between single-motor processivity and mean cargo velocity. The frequency of the rearward travels of the cargo, caused by detachment of the forward motor, should be determined by the motor’s processivity: larger processivity (i.e., lower “off rate”) should lead to decreased frequency of backward travels (Figure 7A, top) and thus higher average velocity (Figure 7A, bottom). Second, increasing load (up to a point) should increasingly favor backward motions relative to forward motions (resulting from more likely detachment of the forward motor). We looked at the ratio of forward motor to rearward motor detachments (Figure 7B) and found that this was true: at no load, the ratio was 1, indicating equal probability of detachment of the “forward” versus “backward” motor, but at a higher load ( $\sim 4$  pN), the forward motor was approximately 3 times as likely to detach. Note that the assumption of equal load sharing made in the steady-state model also prevents this load-induced symmetry breaking (also see Supplemental Data). These rearward motions and corresponding decreases in net cargo velocity are predominantly a low-load effect, for two reasons. First, high load slows down the motors and encourages clustering (Figure S5), so that the magnitude of backward displacements goes down with high load (Figure 7B, bottom). Second, at higher loads, the motors start to share load, so the relative difference in probability of the forward versus backward motor detaching is much less (Figure 7B, top).

Because the velocity change is due to symmetry breaking, i.e., the fact that under low load the forward motor supports most of the load, and thus has a higher probability of detaching, we also expect this effect when motion is opposed by viscous drag (e.g., the cytosol). Model B enabled us to investigate

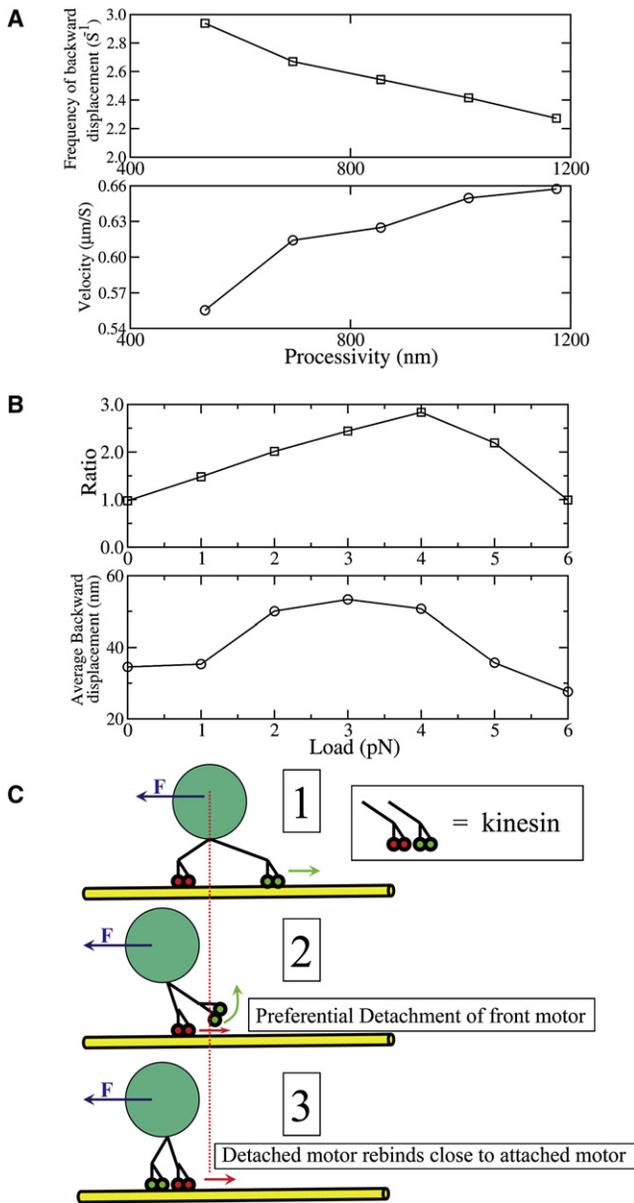


Figure 7. Load-Induced Symmetry-Breaking Results in Altered Mean Velocity at Low Loads

(A) Rate of backward displacements of the cargo, as a function of single-motor processivity (top) and mean cargo velocity as a function of processivity of a single motor (bottom) for  $N = 2$  for applied load of 2 pN. Processivity was calculated with Equation 6 (Supplemental Data) for different values of parameter  $A$ . The values of parameter  $A$  used in the simulation to change the effective single-motor processivity are 67, 87, 107, 127, and 147. Other parameter values are same as in Figure 5A. The simulation was started with the initial condition that both motors are attached randomly to the microtubule.

(B) Ratio of forward to rearward motor detachment events as a function of load (top) and magnitude of backward displacements (after detachment of the forward motor) as a function of load (bottom) for  $N = 2$  motors, linked to the cargo via a 0.32 pN/nm linkage. A ratio of 1 indicates equal probability of detachment of the forward versus rearward motor; ratios higher than 1 indicate that the forward motor is more likely to detach than the rearward motor. The parameter values are the same as in Figure 5A. The simulation was started with the initial condition that both motors are attached randomly to the microtubule. The force-velocity relationship presented in Figures 4A and 4B includes this effect. Note in particular that up to an externally applied load of  $\sim 4.5$  pN ( $\sim 3/4$  max stall for one motor), the two-motor mean

velocity is below the one-motor mean velocity (see Figures 4A and 4B, intersection of red and black curves). When a motor detaches, and subsequently reattaches, the location of reattachment is determined randomly but constrained to be within 110 nm of the cargo, reflecting the motor's length. (C) Cartoon of the sequence of events resulting in small backward displacements of the cargo's center of mass. Initially, in (1), both motors are attached to the microtubule (yellow). Because the load is low, the motors are not clustered, so the location of the cargo's center of mass (dotted red line) is determined predominantly by the forward motor (green), which balances the externally applied load ( $F$ , indicated by blue arrow). Because of the load supported by the forward motor, its processivity is decreased relative to the backward motor, and it has a higher probability of detaching (see ratio, top, [B]). Thus, when one of the two motors detaches, it is more likely to be the forward motor, which results in backward motion of the cargo's center of mass (2), because the externally applied load  $F$  is now supported by the only remaining attached motor (in red). However, because the cargo is held close to the microtubule by the bound motor (in red), the detached motor (in green) now has the opportunity to rebind to the microtubule. When it does (3), the distribution of its new binding locations is determined by the cargo's current center of mass, so that in general, the rebinding of the motor occurs behind its previous attachment location. This completes the cycle, resulting in a small backward motion ([B], bottom) of the cargo's center of mass. Obviously, the more frequent such cycles of detachment/backward motion (determined by a combination of load and the single-motor processivity, [A and B]), the more effect there is on mean cargo velocity.

## Discussion

### Strain Gating and Velocity Sensitivity to Load

It is important to understand how multiple motors function together, because a variety of studies in vivo suggest that cargoes are frequently driven by more than one motor [6], and recent work [6] suggests that regulation of the number of engaged motors could be one important route for regulating transport. Here, we addressed theoretically how two-motor transport differs from one- or three-motor transport. Key predictions of our model agree well with experimental data.

Our model(s) reproduce known single-molecule experimental data (see Figures S1 and S2). They include stochastic single-molecule behavior, as well as thermal fluctuations of the

velocity is below the one-motor mean velocity (see Figures 4A and 4B, intersection of red and black curves). When a motor detaches, and subsequently reattaches, the location of reattachment is determined randomly but constrained to be within 110 nm of the cargo, reflecting the motor's length. (C) Cartoon of the sequence of events resulting in small backward displacements of the cargo's center of mass. Initially, in (1), both motors are attached to the microtubule (yellow). Because the load is low, the motors are not clustered, so the location of the cargo's center of mass (dotted red line) is determined predominantly by the forward motor (green), which balances the externally applied load ( $F$ , indicated by blue arrow). Because of the load supported by the forward motor, its processivity is decreased relative to the backward motor, and it has a higher probability of detaching (see ratio, top, [B]). Thus, when one of the two motors detaches, it is more likely to be the forward motor, which results in backward motion of the cargo's center of mass (2), because the externally applied load  $F$  is now supported by the only remaining attached motor (in red). However, because the cargo is held close to the microtubule by the bound motor (in red), the detached motor (in green) now has the opportunity to rebind to the microtubule. When it does (3), the distribution of its new binding locations is determined by the cargo's current center of mass, so that in general, the rebinding of the motor occurs behind its previous attachment location. This completes the cycle, resulting in a small backward motion ([B], bottom) of the cargo's center of mass. Obviously, the more frequent such cycles of detachment/backward motion (determined by a combination of load and the single-motor processivity, [A and B]), the more effect there is on mean cargo velocity.

cargo, because in many systems involving small numbers of interacting molecules, such fluctuations contribute materially to the overall function of the ensemble. In vivo such fluctuations could be important, because molecular motors are stochastic enzymes, and the  $N = 2, 3,$  or  $4$  motors studied here (and likely driving cargo transport in vivo [6]) are in a small- $N$  limit. By including such single-molecule fluctuations, our models differ from a previous steady-state model [4]. As a first experimental validation of our model, we showed that it correctly predicts the experimentally observed distribution of step sizes of the cargo's center of mass, for a cargo moved by two motors (Figure 2). We also correctly predict the slight subadditivity of motor stalling forces (Figure 3), where the two-motor to one-motor stalling force ratio is  $\sim 1.82$ . For instance, for motors with a single-motor stall of  $\sim 5.7$  pN (Figure 5B, black curve), we predict the two-motor stall to be  $\sim 10.4$  pN (Figure 5B, red curve), yielding a ratio of 1.82, consistent with in vitro experiments (see above). For the continuum model, with the same criteria, the published prediction for the two-motor stall is  $\sim 8.8$  pN [4] (see also red dotted line, Figure 5B), yielding a ratio of 1.54, quite different from the experimental observations in vitro.

Relative to the steady-state model, our model predicts quite different ensemble function: improved ability of multiple motors to transport cargoes under large load, and also more sensitivity of the cargo's velocity to small externally applied loads. Both of these effects directly relate to fluctuations. The velocity sensitivity at low load results from load-induced symmetry breaking, where increased likelihood of detachment of the front motor results in brief backward motions of the cargo's center of mass (Figures 7B and 7C). The predicted decrease in mean velocity of cargoes moved by two versus one kinesin was observed experimentally in vivo (G.T. Shubeita, B.C. Carter, and S.P.G., unpublished data). The velocity sensitivity is a small- $n$  effect: as the number of engaged motors grows, more motors are likely to be in the "vanguard," so even at low loads, cargo fall-backs will become smaller and load sharing will increase. The net result is that for cargoes driven by a high number of motors, we expect no measurable reduction in velocity (relative to single motor velocity) at any load. Indeed, it has already been shown experimentally that in vitro, if very strong drag is not present, there is no correlation between velocity and motor number [16].

These effects were discovered through our study of kinesin, but in retrospect they depend only on relatively generic features of molecular motors. For instance, two fundamental properties give rise to strain gating: first, it requires a relatively stiff motor-cargo linkage, so that when the motor steps forward by a single step, the forward motor ends up supporting a good deal more than  $F/2$ , and second, it requires a generally decreasing force-velocity relationship, so that when the (front) motor is under significant load (relative to the rear motor), it moves through its mechanochemical cycle more slowly, thus allowing the lightly loaded (rear) motor to step first and catch up. Although strain gating will occur under these conditions, the extent that it improves the mean travel of the ensemble will depend on the exact nature of the motor's force-processivity curve—the more difference there is between the high and low load travel distances, the more improvement that results.

The velocity sensitivity at low load is also generic to any motor whose processivity decreases with load. The effect occurs simply because of symmetry breaking: the forward motor feels more load, and therefore is more likely to detach. Although the

magnitude of the effect will depend on a variety of factors, in practice in vivo we see a similar velocity effect for cargoes moved by one versus two dynein motors as well (G.T. Shubeita, B.C. Carter, and S.P.G., unpublished data).

### In Vivo Ramifications

These studies have four implications with regard to transport processes in cells. The first is for cargoes whose motion changes frequently with time, for instance bidirectionally moving cargoes such as virus particles, lipid droplets, mitochondria, and melanophores [17, 18]. In such cargoes, the direction of transport changes every second or so. If motion starts by engaging a single motor, much of the motion will be non-steady state; if they start with a single motor engaged, a large percentage of such cargoes would be expected to fall off the microtubule before reaching steady state (see Figure S14 and Supplemental Data). Thus, in vivo it might be useful to have some method of ensuring that the cargoes remain attached to the microtubules long enough to reach steady state. It is intriguing to speculate that this is one of the roles of proteins like dynactin that provide an independent tether between the microtubule and the cargo.

The second observation is that the mean travel distance of the ensemble is significantly affected by the assumed on rate (below, and not shown). For example, under no load the mean travel distance for two motors is  $\sim 3.4$   $\mu\text{m}$  if the "on" rate is  $5\text{ s}^{-1}$ , but is only  $\sim 2.1$   $\mu\text{m}$  with a  $2\text{ s}^{-1}$  on rate. Thus, for cargoes driven by multiple motors, any mechanism able to tune the on rate could have strong effects on mean transport. Such mechanisms could in principle involve post-translational modifications of the motors themselves, as well as modifications of the microtubules [19, 20] or microtubule-associated proteins [2, 21] that affect the on rate.

The third observation is that the ensemble function is significantly improved if the motors are tightly coupled. Membranes are somewhat flexible, so if the motors are randomly attached to the membrane-bound cargo with some membrane between them, the effective compliance of the system will be quite large, and the motors will not cooperate well. For optimal performance (e.g., to achieve good transport under load), our theoretical results suggest that it is important for motors to be clustered together at a single point on the cargo, and potentially bound to each other via stiff protein linkages. Such clustering has been seen in a number of EM studies [6], though the molecular events leading to such arrangements are currently unknown.

Fourth, over the range of cytosolic drag conditions likely relevant for transport in cells, the effect of changing the number of engaged motors on overall cargo velocity is very different from what is commonly expected (Figure S12). Because it is currently widely believed that because of viscous drag, more motors will move cargoes faster, and the detected 8 nm steps in vivo suggest that many cargoes are indeed moved by  $\sim 1$  motor (see above), these theoretical results based on an experimentally validated theory should have significant implications for interpretation of past and future in vivo experiments.

### Conclusion

We investigated the function of two or more kinesin motors under load and found that the system's performance was strongly affected by the way the motors were linked to the cargo. This suggests that future studies in vivo may need to pay careful attention to motor organization, because alteration

of such organization could play an unappreciated role in determining overall function of the transport system.

#### Experimental Procedures

We started with a single-motor model for kinesin, displaying Michaelis-Menten kinetics, according to the work of [5]. We extended [5] and the subsequently published Monte Carlo model of kinesin [3] to include detachment kinetics of single motors. Complete details of our single-motor model are described in the [Supplemental Data](#) including how it is implemented in a Monte Carlo simulation; there we show that it reproduces known single-molecule *in vitro* experiments.

The Monte Carlo model for multiple motors is an extension of the single-molecule model; the main issue is the geometry for how the motors are attached to the cargo (see [Figure 1](#)). Our multiple-motor model includes the appropriate detachment kinetics when the motor is under super-stall (a force larger than it can move against) as experimentally measured [13]. Further, based on recent work ([15] and M. Lang, personal communication), it assumes that at saturating ATP, a forward load does not affect kinesin's velocity, but does decrease its processivity similar to the effect of a backward-directed force of the same magnitude. Exactly how the simulation is done is discussed in detail in the [Supplemental Data](#).

The steady-state model described was presented by Klumpp et al. [4] and complete details can be found in that manuscript, though we have a brief discussion of the relevant parts in our [Supplemental Data](#).

#### Supplemental Data

Supplemental Data include Supplemental Experimental Procedures and 14 figures and are available at <http://www.current-biology.com/cgi/content/full/18/16/DC1/>.

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